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# **FACESHEET/CORE DISBOND GROWTH IN HONEYCOMB SANDWICH PANELS SUBJECTED TO GROUND-AIR-GROUND PRESSURIZATION AND IN-PLANE LOADING**

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# Overview



- **Background**
- **Road Map**
- **Detailed problem description**
- **Fracture mechanics approach**
  - Development of a test method for fracture toughness testing
  - Finite element modeling
- **Finite element analysis of a panel with circular disbond**
  - Model benchmarking
  - Analysis of a flat panel under internal pressure, in-plane and combined loading
  - Analysis of a curved panel
- **Summary**

# Background



- **Problem**
  - In-service component failures associated with disbonding in unvented honeycomb core sandwich
  - Degradation due to disbonding affects operational safety
  - Failures may discourage use of composites in 'future' vehicles
  - Methods for assessing propensity of sandwich structures to disbonding not fully matured, accepted and documented
  - Methods development is currently being discussed within the Disbond/Delamination Task Group in CMH-17

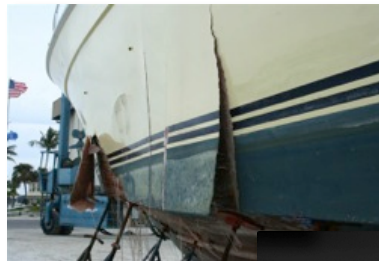
**Space (X-33)**



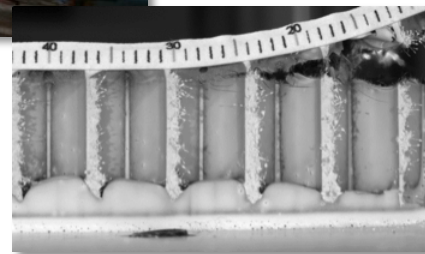
**Detail of flaw**



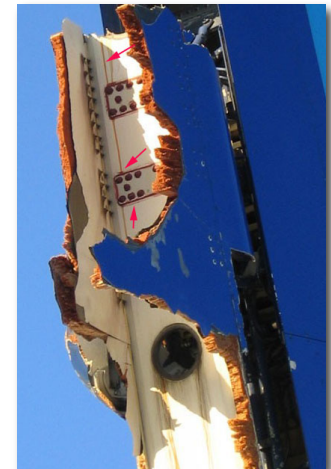
**Marine**



**Fracture test\***



**Aviation\***



\*Focus of this presentation

# Road Map

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- **Ongoing CMH-17/ASTM D30 activity initiated 2012**
- **Current FAA initiative on Continuous Operational Safety (COS)**
- **Objective**
  - **Develop a fracture mechanics based methodology for damage tolerance assessment of sandwich structure**
  - **Assessment of facesheet/core disbonding in sandwich components similar to delamination in composite laminates**
- **Approach**
  - **Coupon test standard development**
    - **Test method for peel-dominated (mode I) interfacial fracture toughness\***
    - **Test method for mode II and mixed-mode interfacial fracture toughness**
  - **Analysis development**
    - **Develop analysis tool for facesheet/core disbonding in sandwich structure\***
    - **Develop models to simulate the ground-air-ground cycle load case\***
  - **Panel testing for analysis validation**
  - **Sandwich disbond methodology development**
  - **Publication**
    - **ASTM D30 fracture toughness standards**
    - **CMH-17 Vol. 6 best practices, guidelines and case studies**

\*Focus of this presentation

# Detailed Problem Description

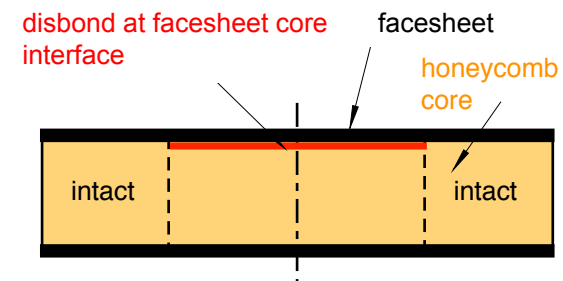


- **Pressure difference between inside and outside of unvented sandwich structures**
  - Caused by alternating changes in ambient pressure and temperature
  - Results in significant deformations and core volume increase
  - Volume increase results in pressure decrease based on the ideal gas law

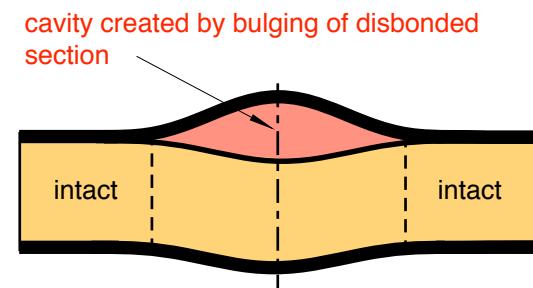
$$p V = n R T$$

- **Initial disbonds between facesheets and core**
  - increase the peeling effect and
  - decrease the structural reliability significantly
- **For an accurate structural analysis, a coupled pressure-deformation problem needs to be solved**

- **Initial configuration at ground elevation**



- **Deformed configuration at cruising altitude**





# Fracture Mechanics Approach – 1 of 2

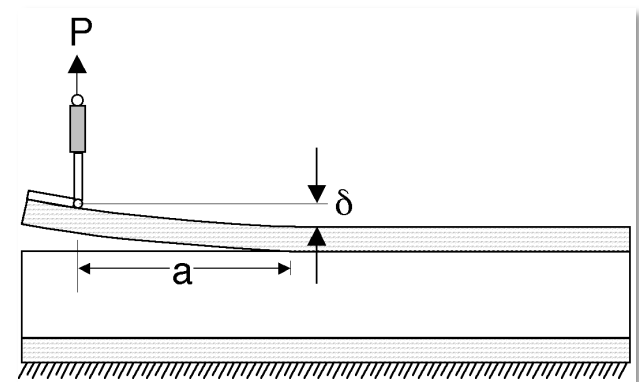
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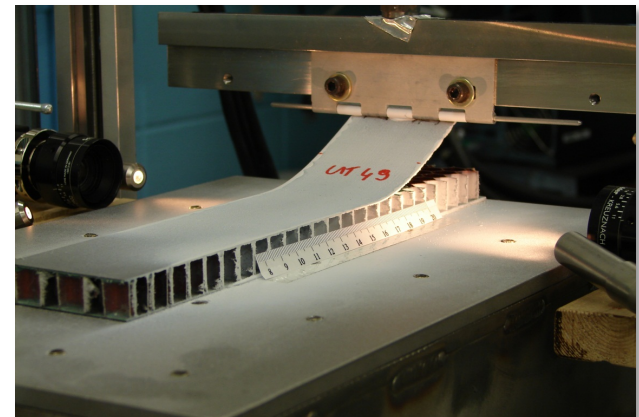
- **Test standard development in ASTM committee D30 (WK 47682)**

- **Characterize properties of facesheet/core interface**
- **Measure fracture toughness  $G_c$**
- **Single cantilever beam (SCB) type configuration was identified as the most appropriate test**
  - Simple loading fixture
  - Disbond front loading is independent of disbond length
  - Disbonding occurs along or near the facesheet/core interface (no kinking into the core)
  - Disbond toughness can be calculated by using a compliance calibration procedure for data reduction
- **Standardized test method for peel-dominated interfacial fracture toughness of sandwich constructions (draft)**
  - Draft includes procedure to determine the SCB specimen dimensions (specimen length, facesheet thickness, initial disbond length)
  - Current round robin activity involves seven research laboratories in the US and Europe

**SCB test schematic**



**Honeycomb sandwich test**



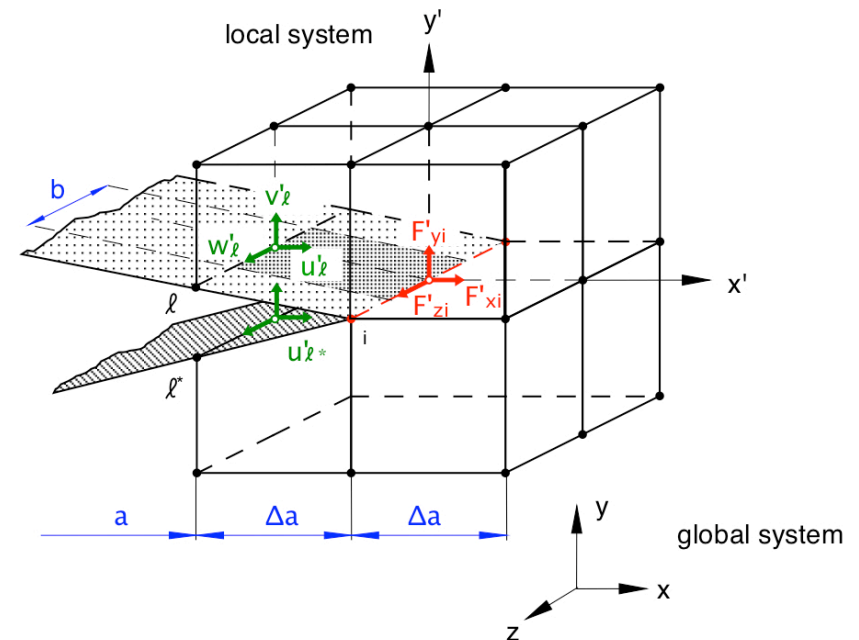
# Fracture Mechanics Approach – 2 of 2



- **Analysis development**

- **Compute the energy release rate along the disbond front**
- **Use the Virtual Crack Closure Technique (VCCT) based on the results obtained from a finite element analysis**
  - Provides mode separation
  - Transformation of nodal forces and displacement into deformed system for non-linear analysis
  - Computation along an arbitrarily shaped delamination path is possible
- **Propagation is predicted to occur once the computed value exceeds the measured fracture toughness**

## Schematic of 3D elements at crack tip



$$G_I = \frac{1}{2\Delta ab} \cdot F'_{yi} \cdot (v'_\ell - v'_{\ell^*})$$

$$G_{II} = \frac{1}{2\Delta ab} \cdot F'_{xi} \cdot (u'_\ell - u'_{\ell^*})$$

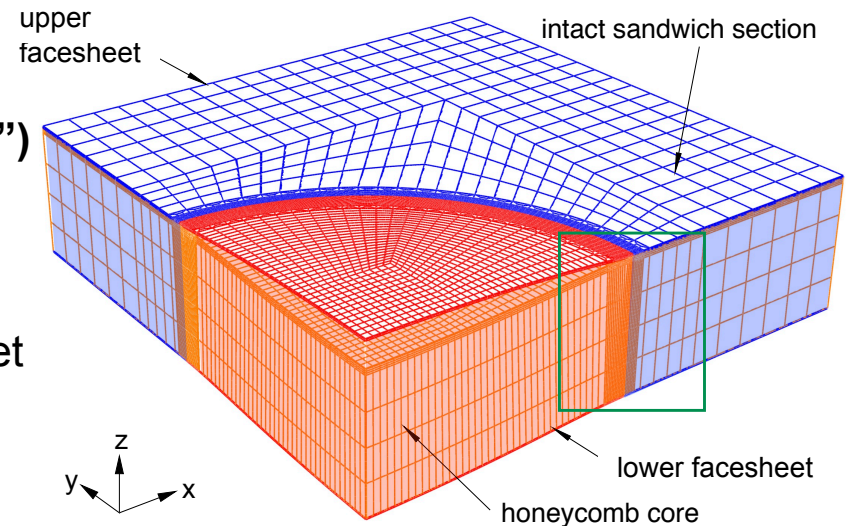
$$G_{III} = \frac{1}{2\Delta ab} \cdot F'_{zi} \cdot (w'_\ell - w'_{\ell^*})$$

# FE Model of a Panel With Disbond – 1 of 4

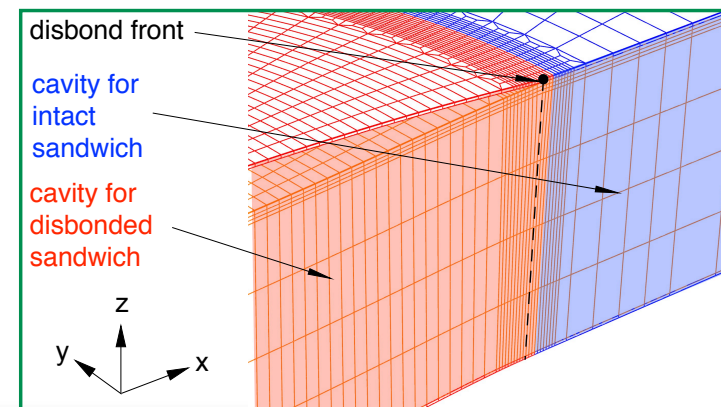


- **A quarter section of a flat panel was modeled**
  - **Circular disbond radius: 152.4 mm (6")**
  - **Square section side dimension: 304.8 mm (12")**
  - **Abaqus/Standard® was used (C3D20 element)**
    - Boundary conditions applied at symmetry planes
    - Surface contact used between top facesheet and core in the disbanded section
- **Sandwich properties**
  - **Thin facesheet: 0.772 mm (0.03")**
    - CYCOM 5320PW plain weave fabric
    - [45/0/90/-45] quasi-isotropic layup
  - **Thick core: 76.5 mm (3.0")**
    - Hexcel HRH-10® honeycomb
    - NOMEX® paper with 48 kg/m<sup>3</sup> (3.0 lb/ft<sup>3</sup>) density and 3.175 mm (1/8") cell size
    - Modeled as an orthotropic, homogeneous continuum

**3D model of a disbanded flat panel**



**Detail near disbond front**



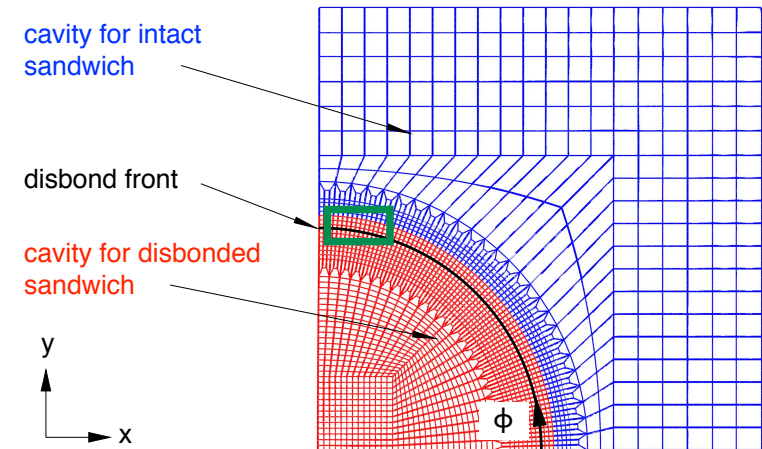


# FE Model of a Panel With Disbond – 2 of 4

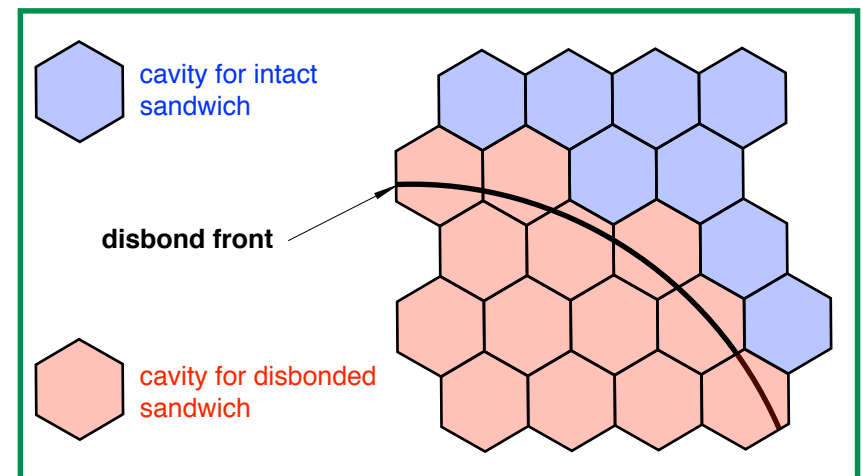


- **Pressure deformation coupling was simulated using fluid-filled cavities**
  - **Abaqus/Standard® feature enabled the definition of fluid-filled cavities enclosed by structural elements**
  - **The ideal gas law is solved within each increment until equilibrium is found**
  - **The volume of the fluid cavities was assumed to be equal to that of the entire sandwich core**
  - **Two separate cavities were defined**
    - One cavity was used to simulate the intact part
    - The other cavity included only the disbonded section
    - The disbonded cavity extended by one cell size, 3.175 mm (1/8”), ahead of the disbond front

**Top view on disbonded flat panel**



**Detail near disbond front**



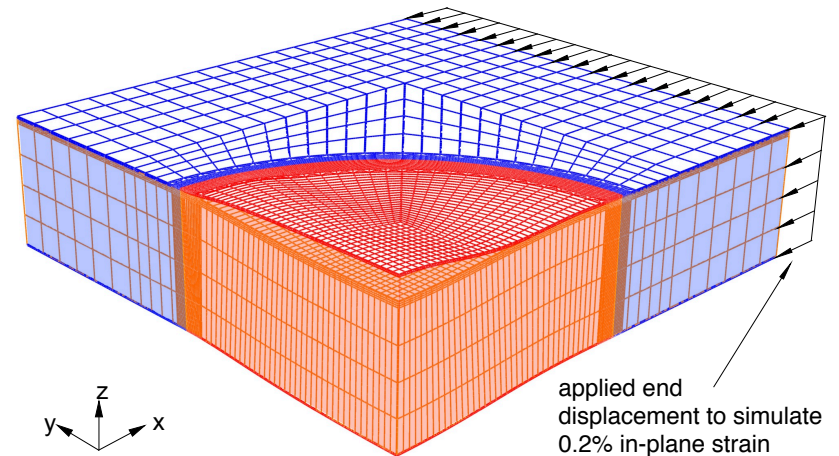
# FE Model of a Panel With Disbond –

## 3 of 4

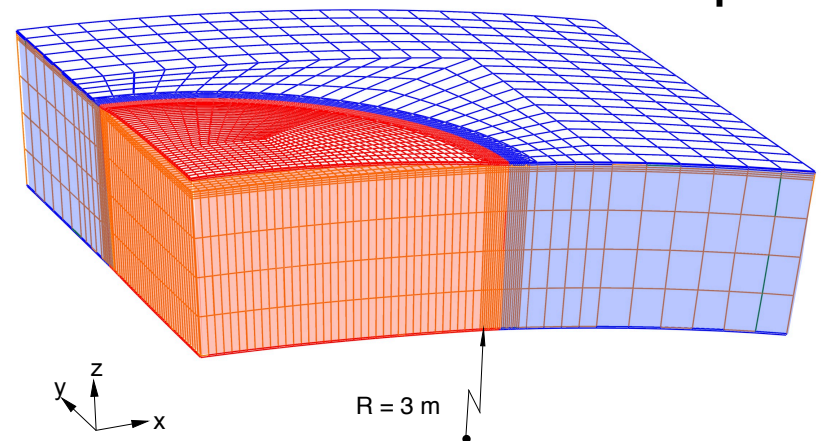


- **Model of a flat panel with in-plane loading**
  - Study the effect of in-plane service load on a flat control surface
  - In-plane displacement applied to the model to simulate a 0.2% (2000  $\mu\epsilon$ ) strain condition during a flight maneuver
  - A compressive strain condition was chosen since it was believed that it would aggravate the tendency to disbond
- **Model of a curved panel**
  - Honeycomb sandwich constructions may be used for cylindrical fuselage structures
  - A 3 m radius (wide body airliner) was chosen for this study

**3D model of a disbonded flat panel**



**3D model of a disbonded curved panel**

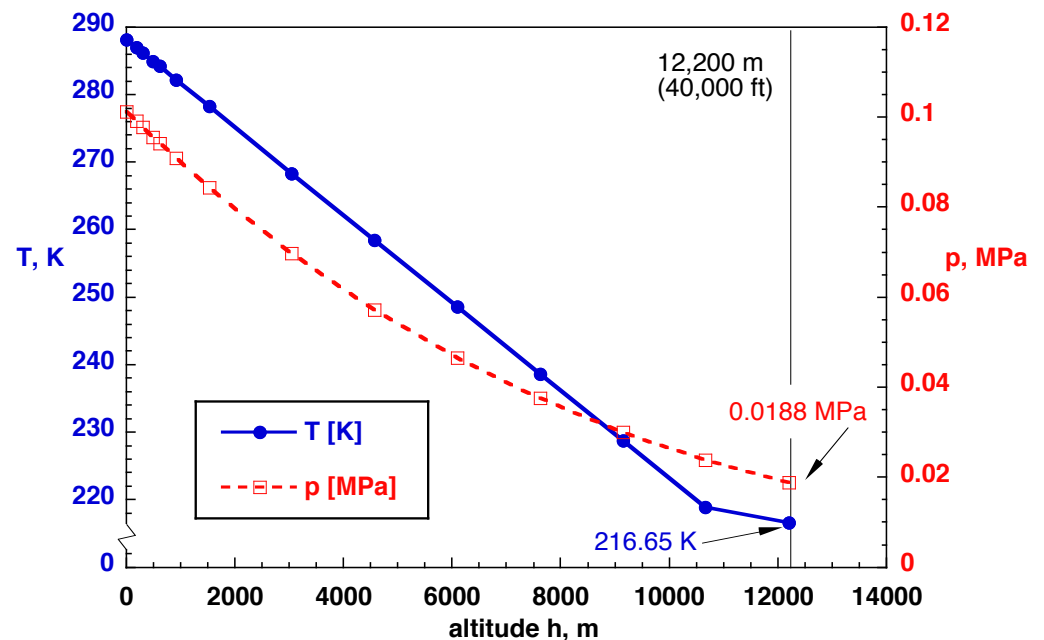


# FE Model of a Panel With Disbond – 4 of 4



- Internal pressurization of the disbond
  - Commercial jetliner ascent scenario was considered from 0 to 12192 m (0 to 40000 ft)
  - The pressure and temperature values were taken from the International Standard Atmosphere ISO 2533
  - The temperature in the core was defined to be equal to the ambient temperature
  - Pressure and volume inside the cavities were calculated during the analysis
- Additional load conditions
  - 0.2% (2000  $\mu\epsilon$ ) strain condition only
  - 0.2% (2000  $\mu\epsilon$ ) strain condition plus GAG cycle

Decrease of temperature and pressure with increasing altitude



# Model Benchmarking – 1 of 3

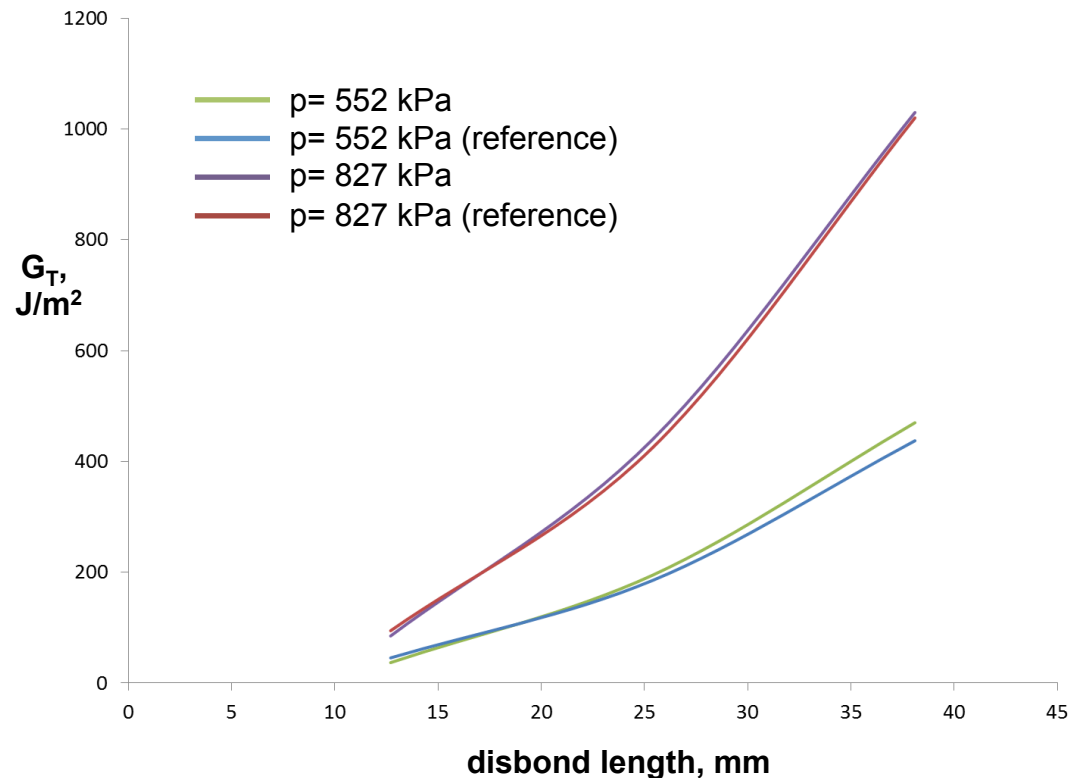


- **X-33 cryogenic fuel tank**
  - **NASA sandwich disbond investigation**
    - Square delamination
    - Panel pressurized by a compressor
    - Defined load, no pressure-deformation coupling
    - Calculations were performed using surface loads
  - **Current analysis approach**
    - Same dimensions as NASA investigation
    - Pressure load case modeled with Abaqus fluid elements
    - VCCT calculation using post-processing routine

- **Result comparison**

- Good correlation between  $G_T$  values calculated using different models

## Energy release rate dependence on disbond length



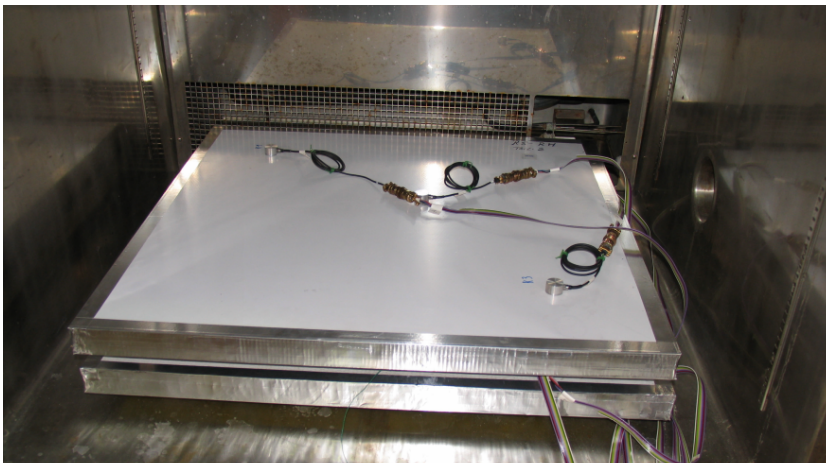


# Model Benchmarking – 2 of 3



- **Sandwich panel with disbond**
  - Panel with 350 mm disbond
  - Pressure-deformation coupling needs to be considered
  - Pressure in disbonded core section was measured during test
  - FE analysis was performed calculating pressure-deformation coupling iteratively

## Airbus test panel in vacuum chamber



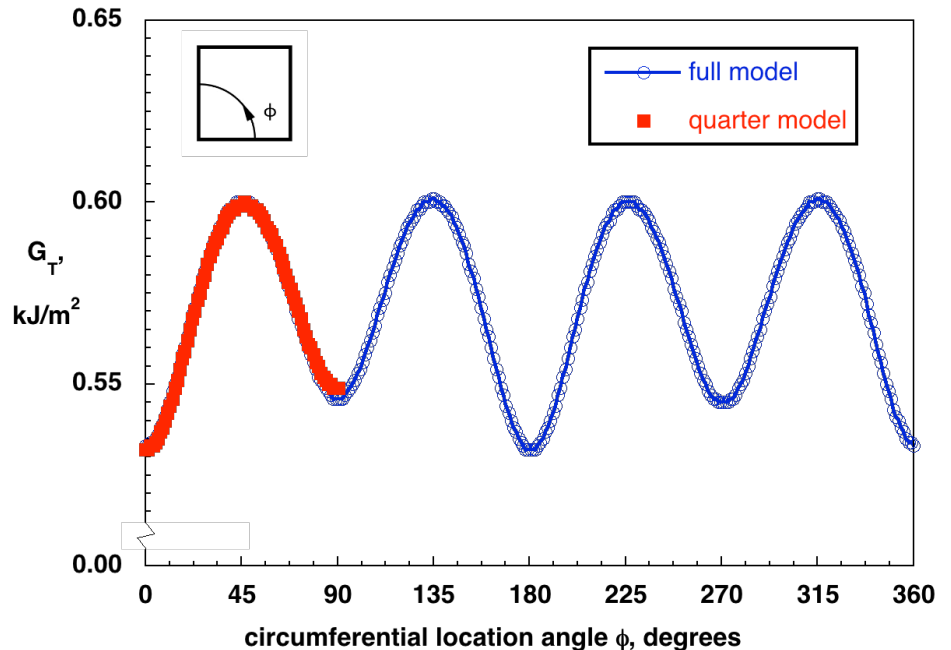
- **Current analysis approach**
  - Same dimensions as Airbus panel
  - Pressure-deformation coupling solved using Abaqus Fluid Cavity Simulation
- **Result comparison**
  - Good correlation for pressure-deformation coupling using different models
  - Pressure in core:
    - Airbus test: 0.0582 MPa
    - Airbus analysis: 0.0577 MPa
    - Current analysis: 0.0571 Mpa
- **Additional validation studies should be performed to compare test results and analysis**
  - Compare deformation field
  - Compare pressure inside the cavity

# Model Benchmarking – 3 of 3



- **Conditions**
  - **12,192 m altitude (40,000 ft)**
    - External pressure  $p=0.0188$  MPa (2.73 lbs/in<sup>2</sup>)
    - External temperature  $T= 216.65$  K (-69.7°F, -56.5°C)
- **Verification for using a FE model of a quarter section of the panel**
  - Excellent agreement of computed  $G_T$  along the front for the currently used quasi-isotropic layup
  - *Deviation, however, for other layups that violate the symmetry conditions of the model*

**Distribution of energy release rate along the disbond front**



# Flat Panel Subjected to Internal Pressure Loading – 1 of 2



- **Parametric study**

- **Variation of**

- Facesheet thickness, number of plies
    - Disbond radius: 50.8 – 762 mm (2.0" – 30.0")
    - Core density: 29 kg/m<sup>3</sup>, 48 kg/m<sup>3</sup>, 80 kg/m<sup>3</sup> (1.8 - 5.0 lb/ft<sup>3</sup>)
    - Core thickness: 12.5 mm, 25.4 mm, 50.8 mm, 76.5 mm (0.5" - 3.0")

- **Results**

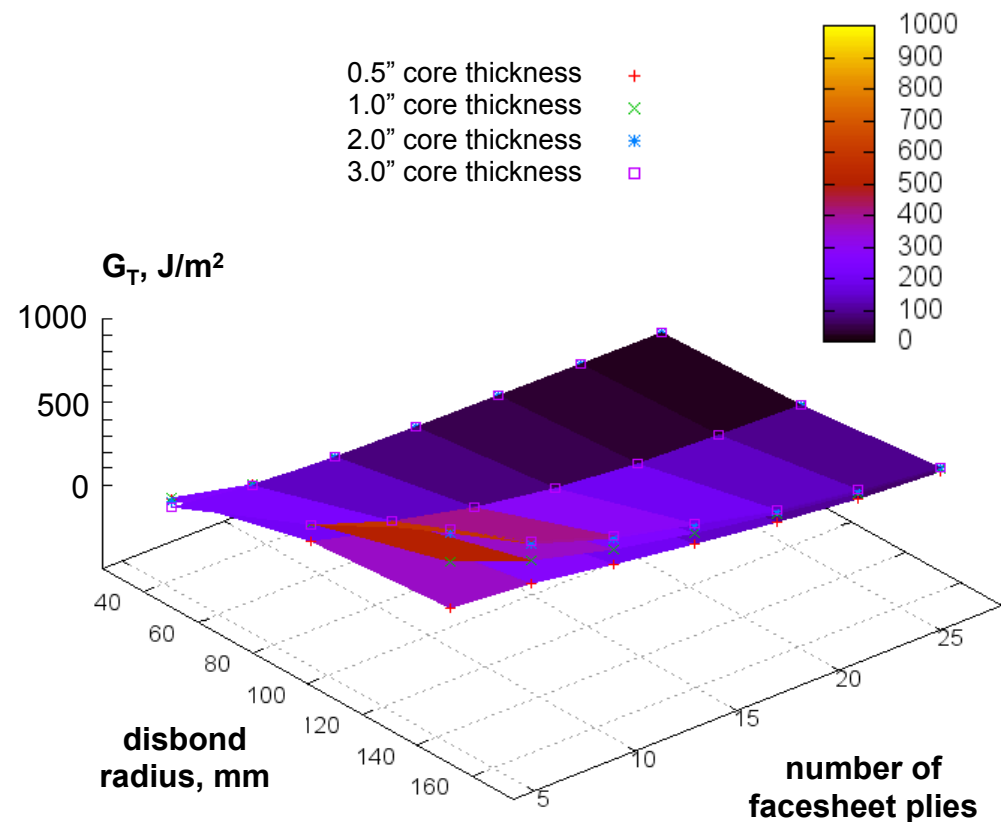
- Variation of core density does not have a significant effect on computed  $G_T$
    - Large disbond radius and thin facesheets result in maximum  $G_T$

- **Following studies**

- Dimensions based on results from parametric study

## Averaged $G_T$ along crack front

3.275 mm (1/8") cell size, 48 kg/m<sup>3</sup> (3.0 lb/ft<sup>3</sup>) core density

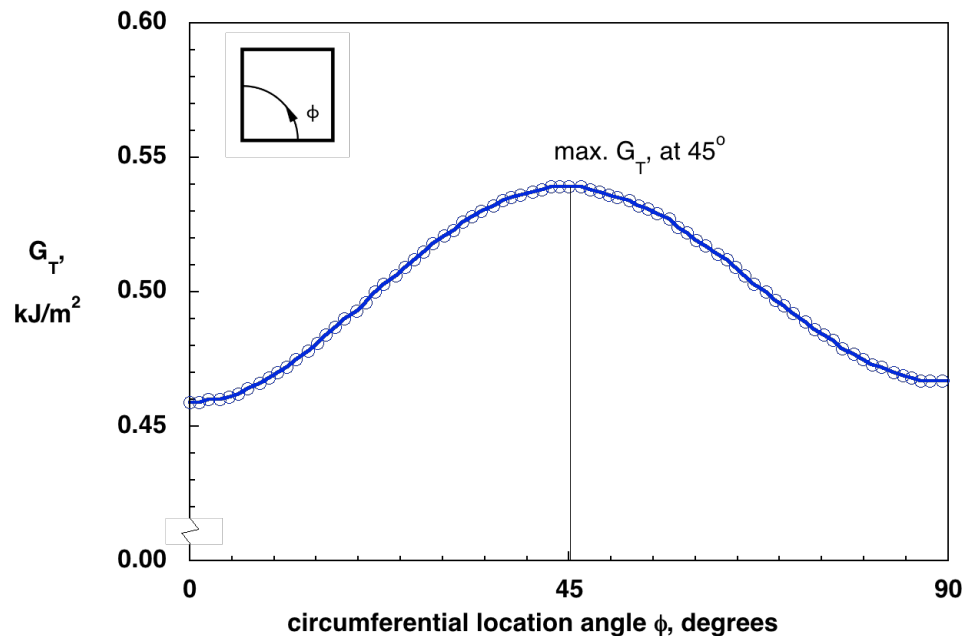


# Flat Panel Subjected to Internal Pressure Loading – 2 of 2



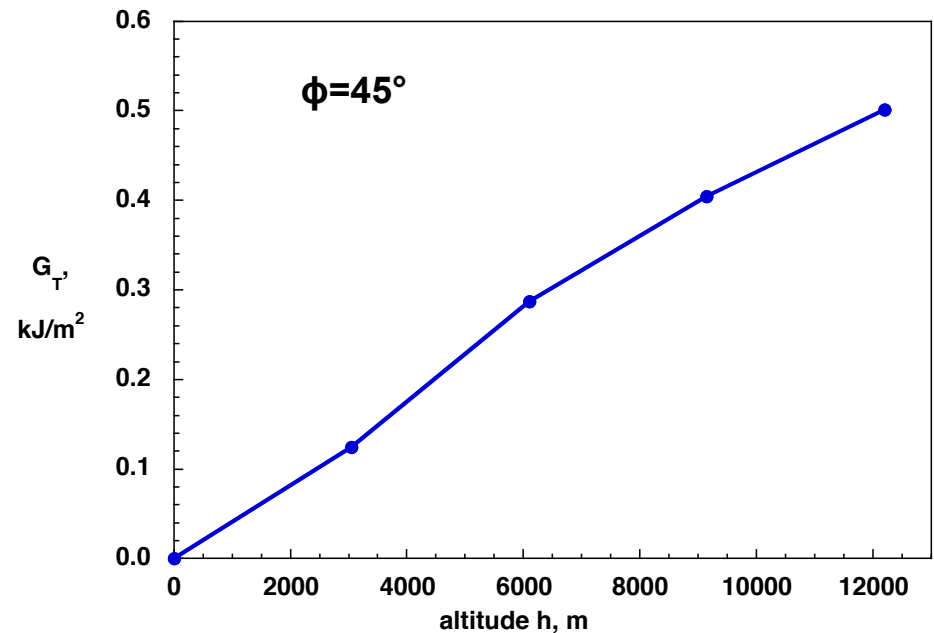
- **Conditions**
  - 12,192 m altitude (40,000 ft)
    - $p=0.0188$  MPa (2.73 lbs/in<sup>2</sup>)
    - $T= 216.65$  K (-69.7°F, -56.5°C)
- **Result**
  - Max  $G_T$  observed at  $\phi=45^\circ$

Energy release rate along the disbond front



- **Conditions**
  - 0 m - 12,192 m altitude
  - Sea level to cruising altitude
- **Results for max  $G_T$  at  $\phi=45^\circ$** 
  - $G_T$  increases monotonically with increasing altitude

Energy release rate dependence on altitude





# Flat Panel Subjected to In-Plane and Combined Loading



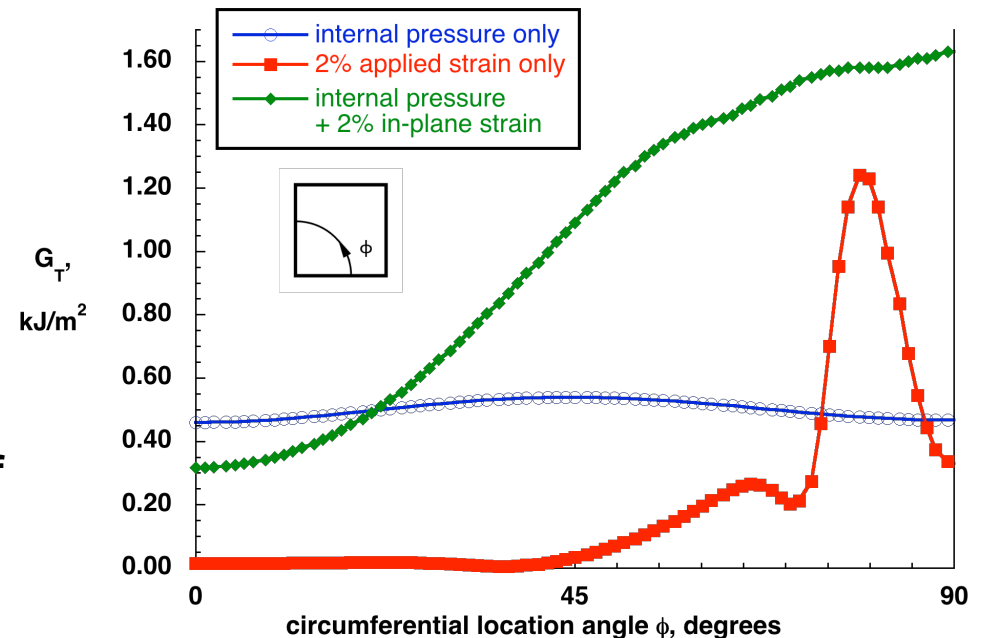
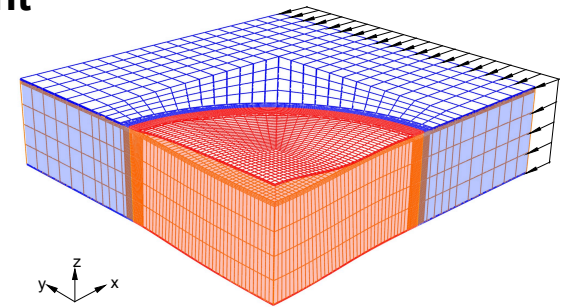
- **Conditions**

- **12,192 m altitude (40,000 ft)**
  - External pressure  $p=0.0188$  MPa
  - External temperature  $T= 216.65$  K
- **0.2% (2000  $\mu\epsilon$ ) applied in-plane strain to simulate service loads on a flat control surface**
- **Combined internal pressure + 0.2% (2000  $\mu\epsilon$ ) in-plane strain**

- **Results**

- Out-of-plane deformation of the disbonded section changes
- Leads to a change in the  $G_T$  distribution
- Addition of in-plane strain leads to an increase in  $G_T$
- Due to non-linearity superposition of the results is not possible

## Distribution of energy release rate along the disbond front



# Analysis of a Curved Panel

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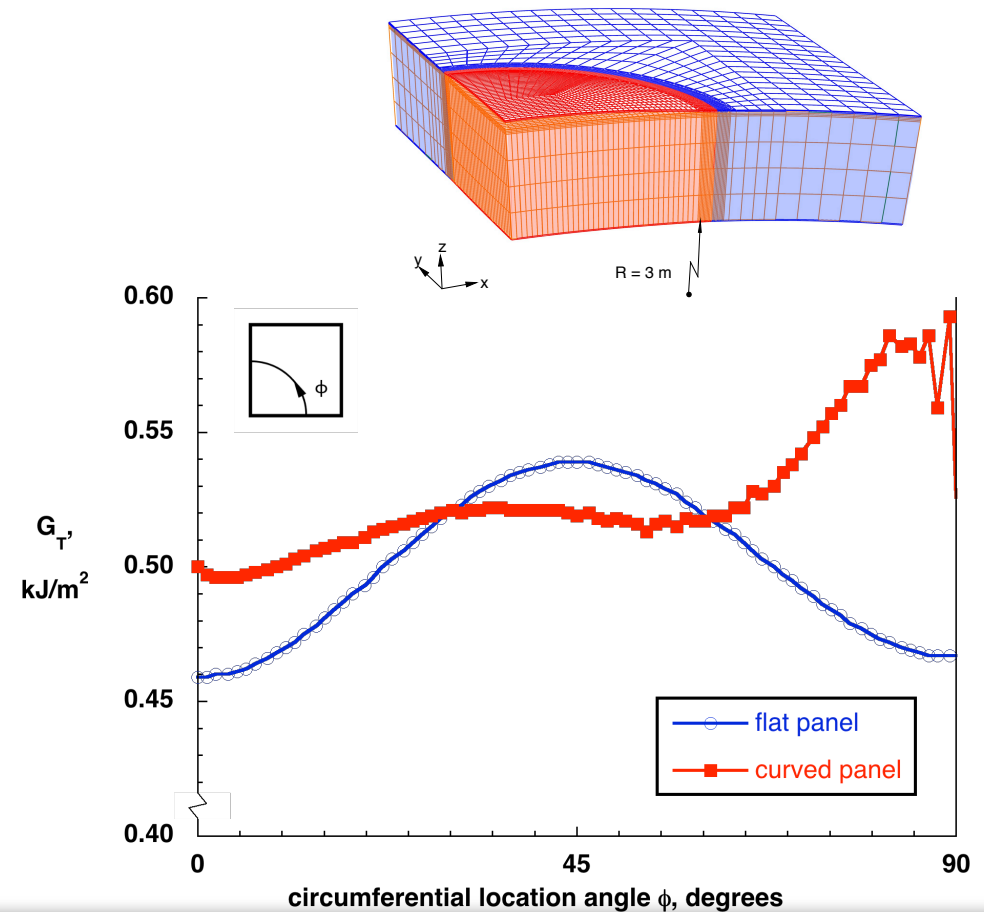
- **Conditions**

- 12,192 m altitude (40,000 ft)
  - External pressure  $p=0.0188$  MPa
  - External temperature  $T= 216.65$  K
- **Flat panel**
- **Curved panel with 3 m radius**

- **Results**

- Symmetry of the  $G_T$  distribution is lost for the curved panel
- Locally and on average the computed  $G_T$  is higher than the result obtained from the flat panel
- Result is unexpected
- In-plane strain may lead to a further increase in computed  $G_T$
- Additional analyses with different radii and more refined mesh should be performed before a definite statement is made

## Distribution of energy release rate along the disbond front



# Summary



- A methodology similar to delamination modeling in composites was developed to assess facesheet/core disbonding in honeycomb sandwich components.
- A sandwich panel containing a circular disbond at the facesheet/core interface was studied using pressure-deformation coupling.
- Large disbonds, thin facesheets, and thick cores are most critical.
- Computed averaged  $G_T$  values increased almost linearly with increasing altitude.
- In-plane compressive strains increased  $G_T$  along the crack front.
- Due to non-linearity of the problem, results for combined load cases cannot be obtained simply by superposition of individual load cases.
- Computed  $G_T$  values were higher for a curved panel than for a flat panel.